

# Scour Downstream of Submerged Parallel Radial Gates

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Authors are congratulated for their paper dealing with investigations of scour downstream of barrage gates in river Nile in Egypt. Similar scour were observed in barrages in India too. Discussor had the opportunity to investigate scour in Farakka barrage on river Ganges and Kosi barrage on river Kosi in India (Mazumder, 2011). Uncontrolled scour and erosion of river bed and banks often result in breaches, meandering and sometimes flow avulsion. Often the safety of barrage itself is threatened due to outflanking.

The main cause of scour downstream of barrages is inadequate energy dissipation in the stilling basin (called basin hereafter) invariably provided downstream of the barrage gates. In all barrages and weirs, there is always flow choking resulting in rise in energy level and afflux upstream of the barrage. The excess energy of flow ( $\Delta E$  as shown in Fig.1 in this discussion paper) above the normal energy level (in pre-barrage state) must be completely dissipated within the basin before the flow moves downstream over natural river bed made of very fine alluvial soil in rivers like Nile and Ganges

In the classical design of the basin, it is presumed that the excess energy ( $\Delta E$ ) is completely dissipated within the basin and the energy of water flowing downstream of the basin will be equal to the normal energy of flow ( $E_2$ ) corresponding to normal flow velocity ( $V_2$ ) at normal tail water depth of flow ( $y_t = d_2$  in Fig. 1). The basin length provided (Fig. 1 in the paper by author) is 150 cm which seems to be more than the length of jump. It is assumed that the basin length is adequate to contain the jump fully. Authors have not mentioned about the basin type adopted as there is no data available for computing pre-jump Froude's Numbers of flow ( $Fr_1$ ) which is found to vary from 2.5 to 8.5 (Figs. 3 to 9 in the paper) for different values of expansion ratio ( $e$ ), head variation ( $h$ ) and jump submergence ( $S_j$ ).

It is well established (Hager, 1992; Peterka, 1958) that the jump becomes steady only when  $Fr_1 > 4.5$ . In most of the barrages in India, pre-jump  $Fr_1$  is found to vary between 2 to 4 - a range in which jump is not perfect. In undular jump ( $1 < Fr_1 < 1.7$ ), weak jump ( $1.7 < Fr_1 < 2.5$ ) and oscillating jump ( $2.5 < Fr_1 < 4.5$ ), energy dissipation is incomplete as the jump is not perfect

Effect of submergence on energy dissipation is well established (Chow, 1973); but the effect of expansion ratio,  $e$  (defined as the ratio between gate opening and channel

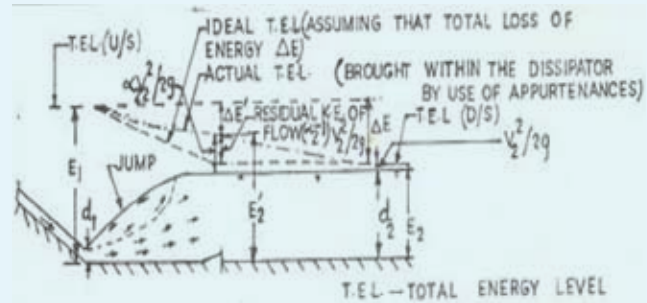


Fig. 1 : Showing Jump, Energy line and Residual Kinetic Energy :  $\Delta E - \Delta E' = (\alpha_2 - 1) V_2^2 / 2g$

width) is investigated by the authors for the first time in the paper. Average scour depth ( $d_s/a_g$ ) is found to increase with expansion ratio ( $e$ ).  $d_s/a_g = 0.24$  when  $e = 1.21$  (for all gates open) compared to  $d_s/a_g = 2.76$  when  $e = 6.12$  (one side gate open) - an unprecedented increase of scour by 1050%. In Fig.1 in the paper, authors gave only a qualitative description of scour depth ( $d_s$ ) and its location ( $l_m$ ). Value of the paper would have further increased if the authors had given the details of scour depths and their locations with respect to the different schedule of gate opening.

As stated earlier, the residual K.E. of flow leaving the basin and expressed as  $(\Delta E - \Delta E')$  (as shown in Fig. 1 in this discussion paper), is responsible for scour downstream of the basin. Higher the residual energy, greater will be the scour. Defining Coriolis' coefficient ( $\alpha$ ) as

$$\alpha = 1 / (AV^3) \int u^3 dA \quad \dots(1)$$

where  $u$  is the the local velocity through an elementary area  $dA$  and  $V$  is the mean velocity of flow over the full cross-sectional area  $A$  of the channel. In uniform flow,  $u = V$  and  $\alpha = 1$ . It may be seen from Fig.1 (in the discussion paper), that the residual K.E. of flow leaving the basin is given by

$$(\Delta E - \Delta E') = (\alpha_2 - 1) V_2^2 / 2g \quad \dots(2)$$

Since  $\alpha$ -value is almost unity at far end downstream where the residual energy is completely dissipated and  $\alpha_2$  is the Coriolis' coefficient at the exit of the basin. The tail water depth ( $y_t = d_2$ ) remains virtually the same after the basin and hence greater the residual K.E. of flow, higher will be the value of  $\alpha_2$ . Only way the excess K.E. of flow can be contained by a flow with same depth ( $y_t$ ) and same mean flow velocity ( $V_2$ ) is through flow non-uniformity resulting in distortion of flow and scour in the tail channel.

Defining efficiency of a basin ( $\eta$ ) as energy dissipator

$$\eta = \Delta E' / \Delta E \quad \dots(3)$$

$$1 - \eta = 1 - (\Delta E' / \Delta E) = (\Delta E - \Delta E') / \Delta E = [(\alpha_2 - 1) V_2^2 / 2g] / \Delta E \quad \dots(4)$$

$$\text{Or, } \eta = 1 - [(\alpha_2 - 1) V_2^2 / 2g] / \Delta E \quad \dots(5)$$

From Eq.(5)  $\alpha_2 = 1$  when  $\eta = 1$  and the basin is 100% efficient as energy dissipator. Eq. (2) and (5) shows that basin efficiency reduces as residual K.E. of flow increases. Since the K.E. of normal flow ( $V_2^2/2g$ ) is a negligible quantity compared to tail water depth ( $d_2 = \eta t$ ), even a small amount of residual K.E. of flow will cause a significant rise in  $\alpha_2$ -value and the basin efficiency will be less.

Discussor (Mazumder,1994; Mazumder and Naresh,1988) performed several experiments in a basin with diverging side walls having 3:1 side splay (I.e.  $e = 1 - L_b/3$ ) where  $L_b$  is the axial length of the basin. In order to improve the basin performance, discussor performed a large number of experiments with several appurtenances e.g. vanes, baffles, bed deflector etc. Without appurtenances, jump front was skewed and the basin performance was extremely poor as indicated by low values of  $\eta$  and high  $\alpha_2$ -values. With the appurtenances in position, basin performance improved remarkably.  $\eta$ -values were nearly 100% and  $\alpha_2$ -values close to unity (Mazumder,2020). Fig. 2 is a plot of basin efficiency ( $\eta$ ) and  $\alpha_2$  for different flows ( $Q$ ).  $\alpha_2$ -values values were computed from velocity distribution measured at the basin end. It may be noticed from Fig.2 that when residual K.E. of flow is only 1% (with  $\eta = 99\%$ ),  $\alpha_2$ -values are 3, 4 and 7 corresponding to  $Q = 31, 15.5$  and  $7.75$  LPS respectively. With 2% residual K.E. ( $\eta = 98\%$ ), the respective values of  $\alpha_2$  were found to be 4, 6 and 12 indicating highly non-uniform velocity.

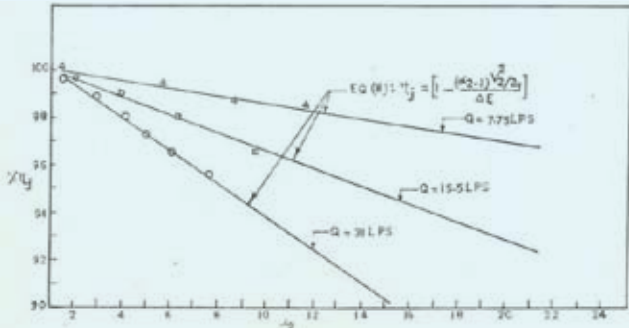


Fig 2 : Showing Jump Efficiency ( $\eta_j$ ) against Coriolis' Coefficient ( $\alpha_2$ )

Discussor developed another innovative method of improving the basin performance by providing adverse slope ( $\beta$ ) to the basin floor.  $\beta$ -value derived by the discussor is given by Eq.(6).

$$\beta = \tan^{-1} [(d_1^2 + d_2^2 + d_1 d_2) \tan \phi / (b d_2 + B d_1 + 2 B d_2 + 2 b d_1)] \quad \dots(6)$$

where,  $b$  and  $B$  are half widths at the entry and exit of basin,  $d_1$  and  $d_2$  are pre-jump and post jump depths respectively and  $\phi$  is the angle of divergence of side walls as shown in

Fig.1 and 3. Large number of experiments were performed by the discussor and the experimental values of  $\beta_{opt}$  was found close to the theoretical values given by Eq. 6. With optimum

slope of basin floor  $\beta_{opt}$  corresponding to the design  $Fr_1$ -value, basin performance improved remarkably with  $\eta$ -values almost 100% and the  $\alpha_2$ -value became almost unity indicating very little residual K.E. leaving the basin (Mazumder,2020). Very high scour occurred with level basin floor and scour was nil when the requisite adverse slope was provided to the basin floor as shown in Photographs 1 and 2.



Photo 1 Showing Scour with  $\beta = 0^\circ$

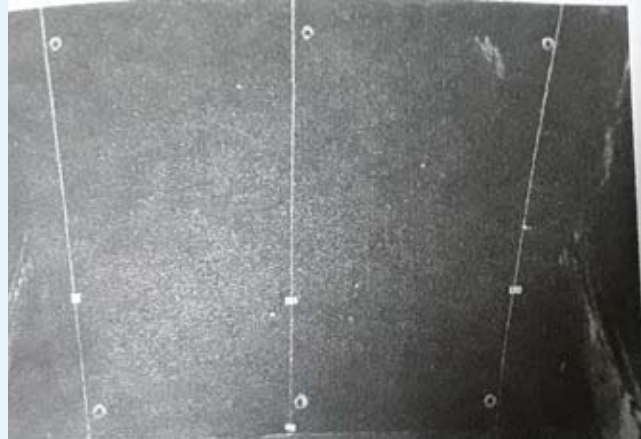


Photo 2 No Scour with  $\beta = \beta_{opt}$

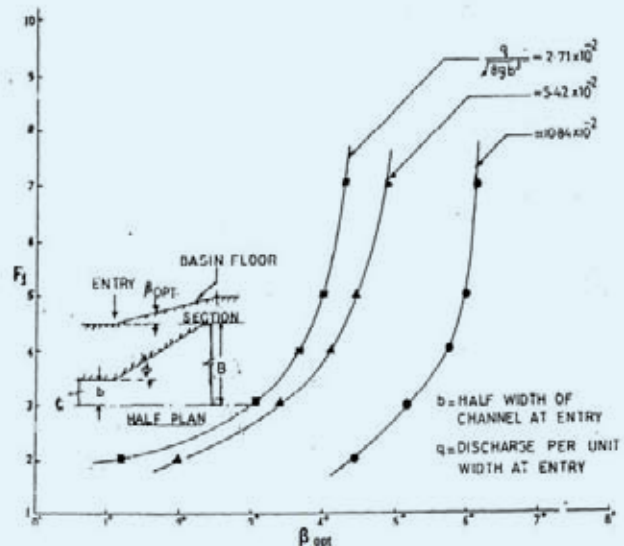


Fig. 3 : Showing Optimum Adverse Slope of Basin Floor ( $\beta_{opt}$ ) Against  $F_1$

It will be appropriate here to analyse the reason why high scour occurs when expansion ratio is high. For that, one must understand the mechanism of energy losses in a hydraulic jump. Prejump K.E. of flow ( $E_1$  in Fig.1) is converted partly to pressure energy ( $d_2$ ). The remaining kinetic energy-transferred to production of turbulence- can not be recovered and is called head loss in a jump. Chaturvedi (1963) and Rouse et al, (1951) measured turbulent quantities in a conical diffuser for finding the head losses in terms of turbulent quantities. Production of turbulence is dependent not only upon pre-Jump  $Fr_1$  but also upon the angle of impact of the incoming and outgoing flow (Hinz...). In a skewed jump with inclined jump front, turbulence production is less compared with that in a jump with impact angle  $00$  to flow axis, found to occur in a classical jump. Authors have given any information neither about the jump front nor about flow conditions downstream with different  $e$ -values as per schedule of gate operation.

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